Experiment – 11 Plate Heat Exchanger

Aim of this Experiment

The Plate Heat exchanger is a 4-plate brazed model that demonstrates the basic principles of heat transfer

Experimental Set – up

The plate heat exchanger is extremely versatile and exhibits one of the most efficient heat exchanger designs in terms of its capacity relative to its volume. In addition, some designs allow on-site variation of capacity by the addition or removal of plate combinations.

The plate heat exchanger finds application in the food processing and chemical industries where different combinations of plates and gaskets can be arranged to suit a particular application.

Brazed construction is used within the refrigeration industry where compact evaporation or condensation of refrigerant is required in association with heating or cooling of some other medium.

The H102B Plate Heat exchanger is a 4-plate brazed model that demonstrates the basic principles of heat transfer. The H102B is designed to be used with the Heat Exchanger Service Module H102.

The heat exchanger is mounted on the panel fascia, supported by two pegs and retained by a releasable cable tie. The 'Hot side' self-sealing plugs are on the left and the 'Cold side' sockets on the right. Once fitted, the H102B may remain in place. Customers in possession of the H102A Concentric Tube Heat Exchanger should fit the H102B Plate first.

The detailed design of the components are shown on the Schematic diagram. Each plate is corrugated to promote turbulence and is perforated to allow the hot and cold streams to remain in sealed passages on opposite sides of the plates and allow the transfer of heat. It is the combination of turbulence, low volume, high surface area and high fluid velocities that give the high heat transfer capacity in a small volume.

In normal operation, hot water from the heater/circulator flows via the 'HOT OUT' braided hose into the upper 'Hot side' coupling. Its temperature at entry to the heat exchanger is measured by a thermocouple sensor **T1**. It then flows through the heat exchanger and leaves from the lower 'Hot side' coupling via the 'HOT RETURN' hose. Its temperature on exit is measured by a similar thermocouple **T2**.

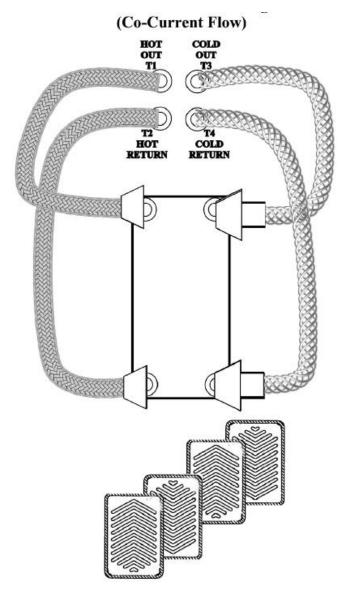
The general arrangement of the hot water flow passages is given in the Flow Direction diagram. Cold water flows through the alternate passage between the alternate plates shown in the Flow Direction diagram. The cold water is fed into the heat exchanger via the 'COLD OUT' reinforced hose and leaves via the 'COLD RETURN' hose.

Thermocouples T3 and T4 that measure the cold water inlet and return temperatures.

The flow direction of the cold stream relative to the hot stream can be reversed by changing the location of cold inlet and exit tubes.



Schematic Representation of Linear Conduction Experiment Unit



Capabilities Of The Plate Heat Exchanger Unit

- 1. To demonstrate indirect heating or cooling by transfer of heat from one fluid stream to another when separated by a solid wall (fluid to fluid heat transfer).
- 2. To perform an energy balance across a plate heat exchanger and calculate the overall efficiency at different fluid flow rates
- 3. To demonstrate the differences between counter-current flow (flows in opposing directions) and concurrent flows (flows in the same direction) and the effect on heat transferred, temperature efficiencies and temperature profiles through a plate heat exchanger.
- 4. To determine the overall heat transfer coefficient for a plate heat exchanger using the logarithmic mean temperature difference to perform the calculations (for countercurrent and co-current flows).
- 5. To investigate the effect of changes in hot fluid and cold fluid flow rate on the temperature efficiencies and overall heat transfer coefficient.
- 6. To investigate the effect of driving force (difference between hot stream and cold stream temperature) with counter-current and co-current flow.

Operating Procedure Of Plate Heat Excganger Unit

Starting

Fit the chosen heat exchanger and connect the Hot/Cold hoses to suit co-current or counter current flow.

Turn on the cooling water supply and open the cooling water flow control valve on the cooling water flowmeter(7). Ensure that cooling water flows freely through the flowmeter and heat exchanger to the drain. Set the cooling water flow to a low value (10-25g/s depending upon water inlet temperature).

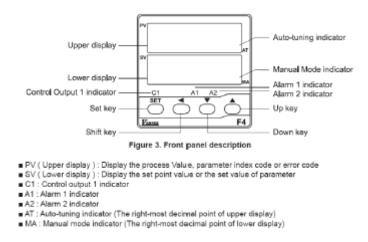
Supply power to the unit and turn on the main switch. The hot water flowmeter(8) should indicate a circulating flow.

Both the digital temperature indicator(6) and the digital water temperature control(5) will illuminate and carry out a self-test before displaying numeric values.

Setting the hot water temperature

The water temperature controller is a digital PID (Proportional Integral Derivative) controller that operates an internal solid state relay which in turn controls power to the 3.0kW water heater. The measured value (Upper display) is sensed at a point close to T1 (Hot Water to Heat Exchanger).

The water temperature controller has the following components.



The Lower Display (shown in the diagram above) shows the SV set point value (the hot water temperature required).

The Upper Display (also shown above) shows Process Value PV (or the ACTUAL temperature of the hot water at the controller sensor). Note that there may be a difference between T1 (Hot Out) temperature and the temperature at the controller sensor. This is normal.

Note that the user has access to the SET, Shift, Up and Down keys. These have different uses.

Setting a New Temperature

To set a new hot water temperature press the Shift key. This will, cause one digit in the Lower display to highlight. The highlighted digit can be changed by using the up and down keys:-

Up Key to increase or Down key to decrease.

This procedure can be repeated for all of the digits in the display by pressing the Shift key repeatedly.

Note that the maximum value that can be set on the H102 unit is 100°C and the maximum normal operating value is 70°C and is limited by the safety cut out at 80°C.

The SET key gives the user access to the following options.

	• • • •		
Display	Description	Range	
P⊻oF	Process value offset correction	-1000~1000 (dP =0000)
		-100.0~100.0 (dP =0000)
		-10.00~10.00 (dP =0000)
		-1.000~1.000 (dP=0000)
oUEL	Control output percentage	0.0~100.0%	
rUn	Control mode	DFF:Off	
		00 : On	
		RE 1:AT1	
		RE2:AT2	
		≂ R∩:Man	

IMPORTANT NOTE that if rUn (the control mode) shows anything OTHER than on, then the PID controller is NOT necessarily controlling the heater.

Under normal running conditions when the set value SV (Lower display) is above the measured value MV the controller will indicate a heating demand by illuminating (or flashing) the white C1 LED Control Output 1 Indicator . When the set value is below the measured value the green C1 LED will be extinguished.

When the heater switch(3) is OFF even if the controller is indicating a heating demand the heater will be off.

Turn on the heater supply switch and if the temperature controller is demanding heat the Red 'HEAT INPUT' neon adjacent to the controller will illuminate or flash.

(Note:- A certain amount of air will come out of solution as the water is heated, but this will be automatically vented).

If this is the first time that the unit has been operated then it may need to run for approximately 15 minutes in order to ensure that the majority of dissolved air is released from the hot water circuit.

Note that there will be a slight difference between T1 and the value indicated on the water temperature controller due to sensor location.

Once the system is stable readings may be taken as required.

Shutting Down

1. Turn off the heater switch.

2. Turn the cooling water flow to a high value, and fully open the hot water flow control valves. 3. When the system has cooled to about $40\Box C$, turn off the mains switch and isolate the unit from the mains.

4. Turn off the cold water supply.

Experiment -11.1

Demonstration of Indirect Heating or Cooling By Transfer of Heat from One Fluid Stream To Another When Separated By A Solid Wall (Fluid To Fluid Heat Transfer).

Aim of This Experiment

This experiment aimed to observe effect of heat transfer form one fluid stream to another one when fluids seperated by a solid wall.

Procedure

The following procedure demonstrates heat transfer from one fluid stream to another when separated by a solid wall.

NOTE that the observations from experiment No 2 may be used for the calculations in thisprocedure in order to save experimental time

Install the Plate Heat Exchanger H102B and connect the cold water circuit to give Counter-Current flow as detailed before.

Turn on the 'MAIN SWITCH' and 'HEATER SWITCH'

Set the hot water temperature controller to 60°C. Set the cold water flow rate Vcold to 15g/sec Set the hot water flow rate V hot to 35g/sec.

Allow the conditions to stabilise and record the observations. The procedure may be repeated with different hot and cold flow rates and different hot water inlet temperature if required.

Sample Test Results

Sample No.	T1	T2	T3	T4	Vhot	V _{cold}
No.						
	°C	°C	°C	°C	G/sec	G/sec
1	51.8	41.5	15.0	37.4	35	15
2						
3						
4						
5						

Experiment -11.2

To perform an energy balance across a plate heat exchanger and calculate the efficiency at different fluid flow rates.

Aim of This Experiment

This experiment aimed to calculate overall heat transfer efficiency in plate heat exchanger at different fluid flow rates.

Procedure

The following procedure demonstrates heat transfer from one fluid stream to another when separated by a solid wall.

NOTE that the observations from experiment No 2 may be used for the calculations in thisprocedure in order to save experimental time

Install the Plate Heat Exchanger H102B and connect the cold water circuit to give Counter-Current flow as detailed before.

Turn on the 'MAIN SWITCH' and 'HEATER SWITCH'

Set the hot water temperature controller to 60°C. Set the cold water flow rate Vcold to 15 g/sec Set the hot water flow rate V hot to 40 g/sec.

Allow the conditions to stabilise and record the observations. The procedure may be repeated with different hot and cold flow rates and different hot water inlet temperature if required.

Sample Test Results

Sample No.	Tl	T2	T3	T4	Vhot	V _{cold}
	°C	°C	°C	°C	G/sec	G/sec
1	51.8	41.5	15.0	37.4	42	16
2						
3						
4						
5						

Experiment -11.3

To demonstrate the differences between counter-current flow (flows in opposing directions) and co-current flows (flows in the same direction) and the effect on heat transferred, temperature efficiencies and temperature profiles through a Plate heat exchanger

Aim of This Experiment

This experiment aimed to demonstrate differences between counter current and co-current flows through a plate heat exchanger

Procedure

The following procedure demonstrates heat transfer from one fluid stream to another when separated by a solid wall.

Install the Plate Heat Exchanger H102B and connect the cold water circuit to give Counter-Current flow as detailed before.

Turn on the 'MAIN SWITCH' and 'HEATER SWITCH'

Set the hot water temperature controller to 60° C. Set the cold water flow rate Vcold to 18 g/sec Set the hot water flow rate V hot to 33 g/sec.

Allow the conditions to stabilise and record the observations. The procedure may be repeated with different hot and cold flow rates and different hot water inlet temperature if required.

Monitor the stream temperatures and the hot and cold flow rates to ensure these too remain close to the original setting. Then record the following: T1, T2, T3, T4, Vhot and Vcold

This completes the basic Counter-Current flow experiment observations.

Next connect the cold water circuit to give **Co-Current flow** as detailed before. Note that there is no need to disconnect the hot water circuit or to turn off the hot water pump during this operation.

Monitor the stream temperatures and the hot and cold flow rates to ensure these remain close to the original setting. Then record the following: T1, T2, T3, T4, Vhot and Vcold

This completes the basic Co-Current flow experiment observations

Sample Test Results

Sample No.	Tl	T2	T3	T4	Vhot	Vcold
	°C	°C	°C	°C	G/sec	G/sec
1	56.1	41.3	14.3	36.2	33	18
2						
3						
4						
5						

Theory of Experiments

Co-current and Counter current flow

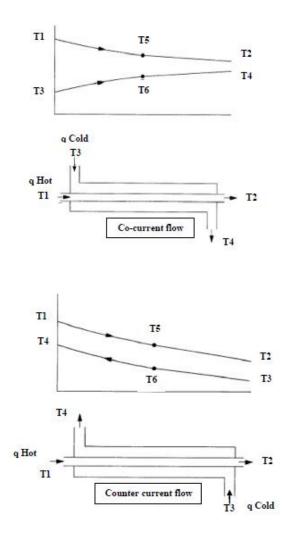
Thermocouples sense the stream temperatures at the four fixed stations: -

- T1 Hot Water INLET to Heat Exchanger
- T2 Hot Water RETURN from Heat Exchanger
- T3 Cooling Water INLET to Heat Exchanger
- T4 Cooling Water RETURN from Heat Exchanger

In addition, two plug-in stations: -

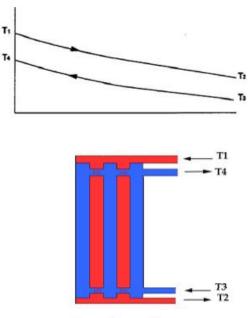
- T5 Hot Mid-position (for Concentric Tube)
- T6 Cold Mid-position (for Concentric Tube)

All thermocouples are duplex sensors, the spare sensor is utilised when HC102A Data Acquisition upgrade is fitted.



A useful measure of the heat exchanger performance is the temperature efficiency.

The temperature change in each stream (hot and cold) is compared with the maximum temperature difference between the two streams. This could only occur in a perfect heat exchanger of infinite size with no external losses or gains.



Counter Current Flow

The temperature efficiency of the hot stream from the above diagram

$$\eta_{Hot} = \frac{T1-T2}{T1-T3} \times 100\%$$

The temperature efficiency of the cold stream from the above diagram

$$\eta_{\text{Cold}} = \frac{\text{T4-T3}}{\text{T1-T3}} \times 100\%$$

The temperature efficiency of the hot stream from the above diagram

$$\eta_{\text{Hot}} = \frac{T1 - T3}{T1 - T4} \times 100\%$$

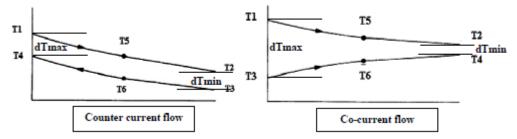
The mean temperature efficiency

$$\eta_{\text{Mean}} = \frac{\eta_{\text{Hot}} + \eta_{\text{Cold}}}{2}$$

As the temperature difference between the hot and cold fluids vary along the length of the heat exchanger, it is necessary to derive a suitable mean temperature difference that may be used in heat transfer calculations. These calculations are not only of relevance in experimental procedures but also of more importance in the design of heat exchangers to perform a particular duty.

The derivation and application of the Logarithmic Mean Temperature Difference (LMTD) may be

found in most thermodynamics and heat transfer textbooks.



The LMTD is defined as

 $LMTD = \frac{dTmax - dTmin}{ln\left(\frac{dTmax}{dTmin}\right)}$

Hence from the above diagrams of temperature distribution for Counter current flow

LMTD =
$$\frac{(T1-T4) - (T2 - T3)}{\ln\left(\frac{(T1-T4)}{(T2 - T3)}\right)}$$

Note that as the temperature measurement points are not fixed on the heat exchanger the LMTD is not the same formula for both Counter-current flow and Co-current flow.

In order to calculate the overall heat transfer coefficient the following parameters must be used with consistent units:

$$U = \frac{\dot{Q}_e}{A \times LMTD}$$

Where

A	Heat transfer area of heat exchanger (m2)
Q e	Heat emitted from hot stream (Watts)
LMTD	Logarithmic mean temperature difference (K)

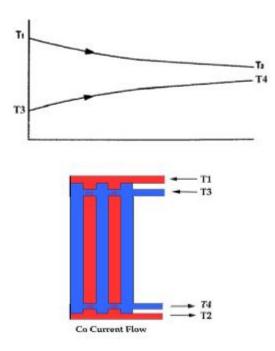
The heat transfer area may be calculated from: -

 $d_{m} = \frac{d_{0} + d_{i}}{2}$ And $A = \pi d_{m} L$

Where

do	Heat transfer tube outside diameter (m)
di	Heat transfer tube inside diameter (m)
dm	Heat transfer tube mean diameter (m)
L	Heat transfer tube effective length (m)

The temperature change in each stream (hot and cold) is compared with the maximum temperature difference between the two streams. This could only occur in a perfect heat exchanger of infinite size with no external losses or gains.



The temperature efficiency of the hot stream from the above diagram

$$\eta_{Hot} = \frac{T1-T2}{T1-T3} \times 100\%$$

The temperature efficiency of the cold stream from the above diagram

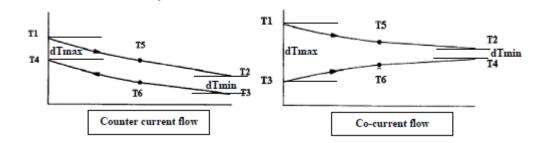
$$\eta_{\text{cold}} = \frac{\text{T4-T3}}{\text{T1-T3}} \times 100\%$$

The mean temperature efficiency

$$\eta_{\text{Mean}} = \frac{\eta_{\text{Hot}} + \eta_{\text{Cold}}}{2}$$

As the temperature difference between the hot and cold fluids vary along the length of the heat exchanger it is necessary to derive a suitable mean temperature difference that may be used in heat transfer calculations. These calculations are not only of relevance in experimental procedures but also more importantly to be used in the design of heat exchangers to perform a particular duty.

The derivation and application of the Logarithmic Mean temperature Difference (LMTD) may be found in most thermodynamics and heat transfer textbooks



The LMTD is defined as

$$LMTD = \frac{dTmax - dTmin}{ln \left(\frac{dTmax}{dTmin}\right)}$$

Hence from the above diagrams of temperature distribution

LMTD =
$$\frac{(T1-T4) - (T2 - T3)}{\ln\left(\frac{(T1-T4)}{(T2 - T3)}\right)}$$

Note that as the temperature measurement points are fixed on the heat exchanger the LMTD is the same formula for both Counter-current flow and Co-current flow.

In order to calculate the overall heat transfer coefficient the following parameters must be used with consistent units:-

$$U = \frac{\dot{Q}_{e}}{A \times LMTD}$$

Where

A Heat transfer area of heat exchanger (m²) Q e Heat emitted from hot stream (Watts) LMTD Logarithmic mean temperature difference (K)

The heat transfer area may be calculated from:-

$$d_m = \frac{d_0 + d_i}{2}$$
And
$$A = \pi d_m L$$

Where

do Heat transfer tube outside diameter (m) di Heat transfer tube inside diameter (m) dm Heat transfer tube mean diameter (m) L Heat transfer tube effective length (m)

Co-Current Flow

For the co-current flow system the calculation procedure is similar but the formulae are as follows

The power emitted from the hot stream Qe

The temperature efficiency of the hot stream from the above diagram

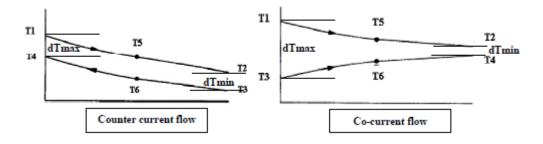
$$\eta_{Hot} = \frac{T1-T2}{T1-T4} \times 100\%$$

The temperature efficiency of the cold stream from the above diagram

$$\eta_{\text{Cold}} = \frac{T4\text{-}T3}{T1\text{-}T3} \times 100\%$$

The mean temperature efficiency

$$\eta_{\text{Mean}} = \frac{\eta_{\text{Hot}} + \eta_{\text{Cold}}}{2}$$



The logarithmic mean temperature difference LMTD (Co-current flow)

LMTD =
$$\frac{(T1-T3) - (T2 - T4)}{\ln\left(\frac{(T1-T3)}{(T2 - T4)}\right)}$$

The Overall heat transfer coefficient U

$$U = \frac{\dot{Q}_{\bullet}}{A \times LMTD}$$

In order to visualise the effect of temperature difference on the overall heat transfer coefficient the calculated data may be plotted against logarithmic mean temperature difference.

For the Hot stream:

For the Cold stream:

 $\dot{Q}a = V_{cold} \rho_{cold} Cp_{Cold} (T4-T3)$ Watts

Appendix – I Symbolas and Units

SYMBOLS AND UNITS

Symbo	1	Units
V _{cold}	Cold stream flow rate	grams s ⁻¹
Vhot	Hot stream flow rate	grams s ⁻¹
T1	Hot fluid inlet temperature	°C
T2	Hot fluid outlet temperature	°C
T3	Cold fluid inlet temperature	°C
T4	Cold fluid outlet temperature	°C
Δt_{hot}	Decrease in hot fluid temperature	К
Δt_{Cold}	Increase in cold fluid temperature	К
dT hot	Decrease in hot fluid temperature	К
dT cold	I Increase in cold fluid temperature	К
Ν	Number of heat transfer plates with hot and cold fluid on both sides	-
T _{mean}	Mean temperature	°C
ρ	Density of stream fluid	kg litre
Ср	Specific Heat of stream fluid	kJkg ⁻¹ K ⁻¹
Q e	Heat flow rate from hot stream	Watts
Q a	Heat flow rate to cold stream	Watts
Q f	Heat loss to surroundings	Watts
LMTD	Logarithmic mean temperature difference	К
F	LMTD correction factor for plate heat exchanger	-
Α	Heat transfer surface area	m^2
a	Projected area of each heat transfer plate	m^2
U	Overall heat transfer coefficient	Wm ⁻² K ⁻¹
ηThermal	Thermal efficiency	%
η _{Thermal} η _{hot}	Thermal efficiency Temperature efficiency hot stream	% %
	-	
η_{hot}	Temperature efficiency hot stream	%
η_{hot} η_{cold} η_{mean}	Temperature efficiency hot stream Temperature efficiency cold stream	%
η _{hot} η _{cold} η _{mean} dTmax	Temperature efficiency hot stream Temperature efficiency cold stream Mean temperature efficiency	% % %

Appendix – II Some Useful Data

PLATE HEAT EXCHANGER H102B

Plate Material	316 Stainless steel
Plate overall dimensions	0.072m x 0.189m
Total heat transfer area	0.024m ²
Number of plates	4
Number of channels:	
Hot side	2
Cold side	3
Weight – Full	0.846kg
Weight – empty	0.776kg

Table 1 Specific Heat capacity Cp of Water in kJ kg ⁻¹

-	0	1	2	3	4	5	6	7	8	9
0	4.1274	4.2138	4.2104	4.2074	4.2054	4,2019	4.1996	4.1974	4.1954	4.1936
10	4.1919	4.1904	4.189	4.1877	4.1866	4.1855	4.1864	4.1837	4.1829	4.1822
20	4.1816	4.181	4.1805	4.1801	4.1797	4.1793	4.1790	4.1787	4.1785	4.1783
30	4.1782	4.1781	4.1780	4.1780	4.1779	4.1779	4.1780	4.1780	4.1781	4.1782
40	4.1783	4.1784	4.1786	4.1788	4.1789	4.1792	4.1794	4.1796	4.1799	4.180
50	4.1804	4.1807	4.1811	4.1814	4.1817	4.1821	4.1825	4.1829	4.1833	4.1837
60	4.1841	4.1846	4.1850	4.1855	4.1860	4.1865	4.1871	4.1876	4.1882	4.1887
70	4.1893	4.1899	4.1905	4.1912	4.1918	4.1925	4.1932	4.1939	4.1964	4.1954
Tou	se the tabl	e the verti	ical colum	ms denote	whole de	grees and	the Horiz	ontal rows	denote te	ms of

To use the table the vertical columns denote whole degrees and the Horizontal rows denote tens of degrees. For example the bold value 4.1792 kJ kg-1 is at 40 + 5 = 45 °C.

Alternatively the equation Cp = $6 \times 10^{-9} t^4 - 1.0 \times 10^{-6} t^3 + 7.0487 \times 10^{-5} t^2 - 2.4403 \times 10^{-3} t + 4.2113$ may be used if the data is to be calculated using a spreadsheet.

Table 2 Density of Water in kg Litre⁻¹

1 40	Table 2 Delisity of Water III kg Litte								
·C	0	2	4	6	8				
0	0.9998	0.9999	0.9999	0.9999	0.9999				
10	0.9997	0.9995	0.9992	0.9989	0.9986				
20 30	0.9982	0.9978	0.9973	0.9968	0.9962				
40	0.9957	0.9950	0.9944	0.9937	0.9930				
50	0.9922	0.9914	0.9906	0.9898	0.9889				
60	0.9880	0.9871	0.9862	0.9852	0.9842				
70	0.9832	0.9822	0.9811	0.9800	0.9789				
	0.0110	9.2 C 99	A		0.0100				

To use the table the vertical columns denote degrees and the Horizontal rows denote tens of degrees. For example the bold value 0.9906 kg is at 40 + 4 = 44 °C.

Alternatively the equation $\rho = -4.582 \times 10^{-6} t^2 - 4.0007 \times 10^{-5} t + 1.004$ may be used if the data is to be calculated using a spreadsheet.